

Strong dissipation feedback: stabilisation of mixed ODE-PDE port-Hamiltonian systems

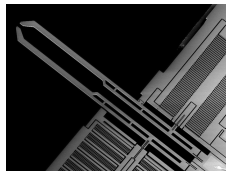
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Motivation



- 1 **Model** that includes the **rigid** and the **flexible** dynamics
- 2 **Control law** that **moves** the system in the desired configuration and **stabilizes** the flexible deformations

Motivation

Models' possible dynamics:

- Actuator inertia dynamics
→ ODE
- Rigid dynamics → ODE
- Flexible dynamics → PDE

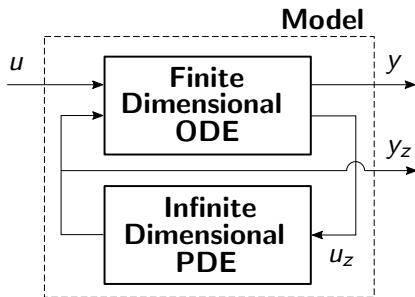


Figure: Model composed by the interconnection between ODE and PDE with **input on the ODE**.

Motivation

Models' possible dynamics:

- Actuator inertia dynamics
→ ODE
- Rigid dynamics → ODE
- Flexible dynamics → PDE

We **restrict** our study on:

- Boundary control systems
- Linear models
(Approximation)
- First order PDE

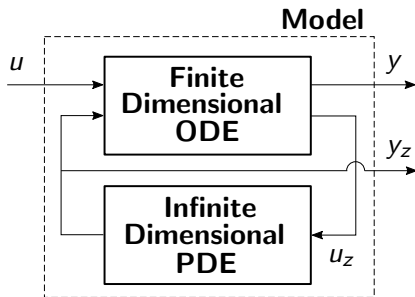


Figure: Model composed by the interconnection between ODE and PDE with **input on the ODE**.

Motivation

We focus on the **port-Hamiltonian** (PH) framework.

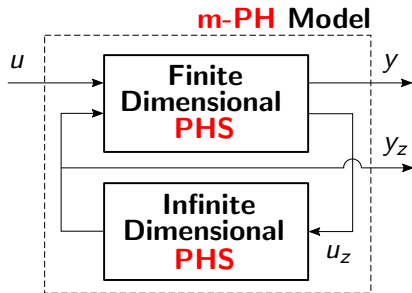


Figure: Mixed port-Hamiltonian (m-PH) system.

Motivation

We focus on the **port-Hamiltonian (PH)** framework.

We restrict our analysis to *models without internal dissipation*.

Therefore:

- Energy is a conserved quantity
- Dissipation can be added only through the control law

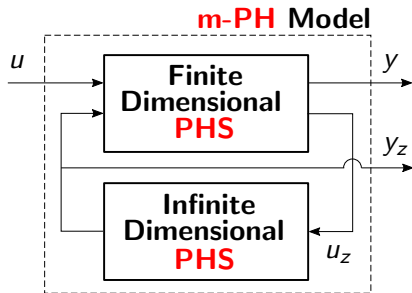


Figure: Mixed port-Hamiltonian (m-PH) system.

Overview

- 1 Introduction
- 2 Strong dissipation feedback
- 3 Strong dissipation & position control
- 4 Comparison of the control laws
- 5 Conclusions, ConFlex fellowship & future plans

Introduction

Introduction

Let $z \in L_2([0, L], \mathbb{R}^n)$, $p \in \mathbb{R}^m$ and consider the following m-pH system

$$\begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} = \begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -C_1 \mathcal{H} & 0 \end{bmatrix} \begin{bmatrix} z \\ p \end{bmatrix} + \begin{bmatrix} 0 \\ u \end{bmatrix}$$

$$y = M^{-1} p$$

with the properties,

- $\mathcal{H} \in C^1([0, L]; \mathbb{R}^{n \times n})$, $\mathcal{H}(\xi)$ is self adjoint for all $\xi \in [0, L]$ and $cI \leq \mathcal{H}(\xi) \leq CI$ for all $\xi \in [0, L]$ and some $C, c > 0$ independent of ξ
- $P_1 \in \mathbb{R}^{n \times n}$ is invertible and self adjoint, $P_0 \in \mathbb{R}^{n \times n}$ is skew adjoint
- $M \in \mathbb{R}^{m \times m}$ is $M > 0$ and symmetric

Introduction

Define the extended state $x = [z \ p]^T \in X = L_2([0, L], \mathbb{R}^n) \times \mathbb{R}^m$

$$\begin{aligned} \begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} &= \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -C_1 \mathcal{H} & 0 \end{bmatrix}}^{A_m} \begin{bmatrix} z \\ p \end{bmatrix} + \begin{bmatrix} 0 \\ u \end{bmatrix} \\ y &= M^{-1} p \end{aligned}$$

$$D(A_m) = \{x \in X \mid z \in H^1, B_1(\mathcal{H}z) = M^{-1}p, B_2(\mathcal{H}z) = 0\}.$$

Input output operators, with $u_z = B_1(\mathcal{H}z)$ and $y_z = C_1(\mathcal{H}z)$

$$\begin{aligned} B(\mathcal{H}z) &= \begin{bmatrix} B_1(\mathcal{H}z) \\ B_2(\mathcal{H}z) \end{bmatrix} = \begin{bmatrix} W_{B,1} \\ W_{B,2} \end{bmatrix} \begin{bmatrix} \mathcal{H}(0)z(0, t) \\ \mathcal{H}(L)z(L, t) \end{bmatrix} \\ C(\mathcal{H}z) &= \begin{bmatrix} C_1(\mathcal{H}z) \\ C_2(\mathcal{H}z) \end{bmatrix} = \begin{bmatrix} W_{C,1} \\ W_{C,2} \end{bmatrix} \begin{bmatrix} \mathcal{H}(0)z(0, t) \\ \mathcal{H}(L)z(L, t) \end{bmatrix} \end{aligned} \quad (1)$$

Control Design & Stability Analysis

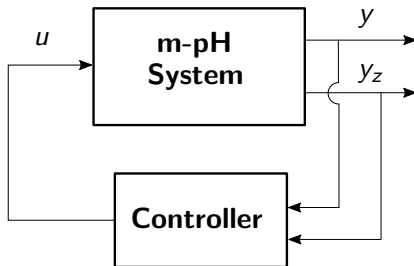


Figure: Control problem of a class of m-PH system.

Control Design & Stability Analysis

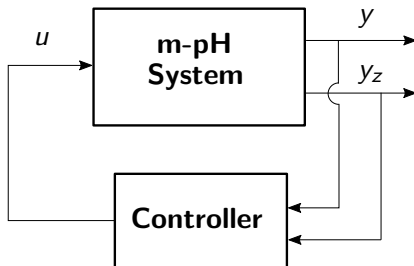


Figure: Control problem of a class of m-PH system.

Control objectives using y and y_z :

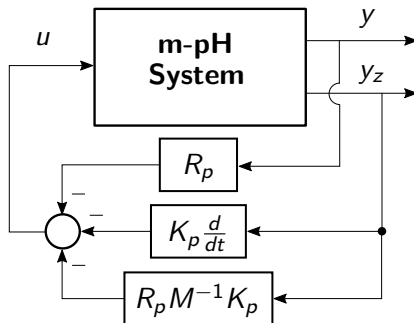
- 1 Stabilise the system to the origin, *i.e.* the state $x = [z \ p]^T$ has to be such that $x(t) \xrightarrow[t \rightarrow \infty]{} 0$
- 2 Stabilise the system to a desired position, *i.e.* the state $x = [z \ p \ q]^T$ has to be such that $x(t) \xrightarrow[t \rightarrow \infty]{} 0$

Strong dissipation feedback

Strong dissipation feedback: Exponential stability

Consider $R_p, K_p \in \mathbb{R}^{m \times m}$ diagonal and positive definite

$$u = \underbrace{-R_p M^{-1} p}_{\text{Classical dissipation}} \underbrace{-K_p \frac{d}{dt}(C_1(\mathcal{H}z))}_{\text{Strong dissipation}} \underbrace{-R_p M^{-1} K_p C_1(\mathcal{H}z)}_{\text{Additional term}} \quad (2)$$



Strong dissipation feedback: Exponential stability

Closed-loop operator

$$\begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} = \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -(I + R_p M^{-1} K_p) C_1 \mathcal{H} - K_p \frac{d}{dt} C_1 \mathcal{H} & -R_p M^{-1} \end{bmatrix}}^{A_1} \begin{bmatrix} z \\ p \end{bmatrix}$$

$$D(A_1) = \{x \in X \mid z \in H^1, \mathcal{B}_1(\mathcal{H}z) = M^{-1}p, \mathcal{B}_2(\mathcal{H}z) = 0\}$$

Strong dissipation feedback: Exponential stability

Closed-loop operator

$$\begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} = \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -(I + R_p M^{-1} K_p) C_1 \mathcal{H} - K_p \frac{d}{dt} C_1 \mathcal{H} & -R_p M^{-1} \end{bmatrix}}^{A_1} \begin{bmatrix} z \\ p \end{bmatrix}$$

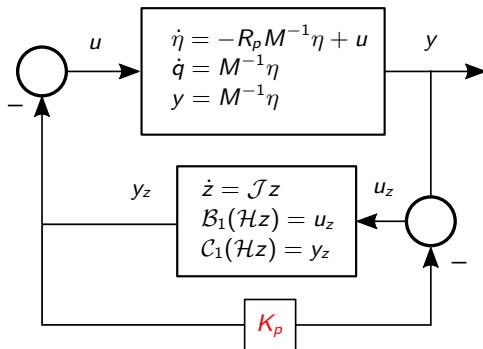
$$D(A_1) = \{x \in X \mid z \in H^1, B_1(\mathcal{H}z) = M^{-1}p, B_2(\mathcal{H}z) = 0\}$$

Change of variables $\eta = p + K_p C_1(\mathcal{H}z)$

$$\begin{bmatrix} \dot{z} \\ \dot{\eta} \end{bmatrix} = \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -C_1 \mathcal{H} & -R_p M^{-1} \end{bmatrix}}^A \begin{bmatrix} z \\ \eta \end{bmatrix}$$

$$D(A) = \{x \in X \mid z \in H^1, B_1(\mathcal{H}z) = M^{-1}(\eta - K_p C_1(\mathcal{H}z)), B_2(\mathcal{H}z) = 0\}$$

Strong dissipation feedback: Exponential stability



- ① Classical finite and infinite dimensional port-Hamiltonian systems
- ② Boundary dissipation on the infinite dimensional system

Strong dissipation feedback: Exponential stability

Theorem 1

The operator A generates a contraction C_0 -semigroup in the space $X = L_2([0, L], \mathbb{R}^n) \times \mathbb{R}^m$ equipped with the inner product

$$\langle x_1, x_2 \rangle = \langle z, \mathcal{H}z \rangle_{L_2} + \eta^T M^{-1} \eta. \quad (3)$$

Theorem 2

Assume that the input $u_z = \mathcal{B}_1(\mathcal{H}z)$ and output $y_z = \mathcal{C}_1(\mathcal{H}z)$ are such that

$$\|\mathcal{H}z(0, t)\|^2 \leq \|u_z(t)\|^2 + \|y_z(t)\|^2$$

or

$$\|\mathcal{H}z(L, t)\|^2 \leq \|u_z(t)\|^2 + \|y_z(t)\|^2$$

then the state trajectory $x(t)$ generated by the closed-loop operator A is such that $\|x(t)\| \leq M_{w_0} e^{-w_0 t}$, where $M_{w_0}, w_0 > 0$.

Strong dissipation feedback: Exponential stability

Sketch of the proof.

We consider the Lyapunov function $V = \frac{1}{2}\langle z, \mathcal{H}z \rangle + \frac{1}{2}\eta^T M^{-1}\eta$,

$$\dot{V} = \langle x, Ax \rangle = -y_z^T K_p y_z - (M^{-1}\eta)^T R_p (M^{-1}\eta). \quad (4)$$

It is possible to find a $c(t) > 0$ for all $t > 0$, such that

$$V(t) \leq \frac{1}{1 + c(t)} V(0). \quad (5)$$

Since the Lyapunov function is equivalent to the state norm, we get

$$\|x(t)\|^2 \leq \frac{1}{1 + c(t)} \|x(0)\|^2. \quad (6)$$

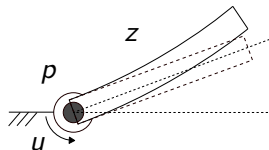
Hence, the semigroup $T(t)$ ($x(t) = T(t)x_0$) is such that $\|T(t)\| < 1$, and therefore there exist $M_{w_0}, w_0 > 0$ such that $\|x(t)\| \leq M_{w_0} e^{-w_0 t}$.

Strong dissipation & position control

Strong dissipation & position control: Asymptotic stability

Example: flexible beam clamped on a rotating inertia
port-Hamiltonian representation

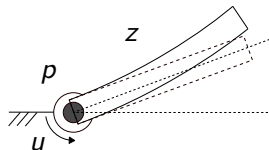
$$\begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} = \begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -C_1 \mathcal{H} & 0 \end{bmatrix} \begin{bmatrix} z \\ p \end{bmatrix} + \begin{bmatrix} 0 \\ u \end{bmatrix}$$



Strong dissipation & position control: Asymptotic stability

Example: flexible beam clamped on a rotating inertia
port-Hamiltonian representation

$$\begin{bmatrix} \dot{z} \\ \dot{p} \end{bmatrix} = \begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 \\ -C_1 \mathcal{H} & 0 \end{bmatrix} \begin{bmatrix} z \\ p \end{bmatrix} + \begin{bmatrix} 0 \\ u \end{bmatrix}$$



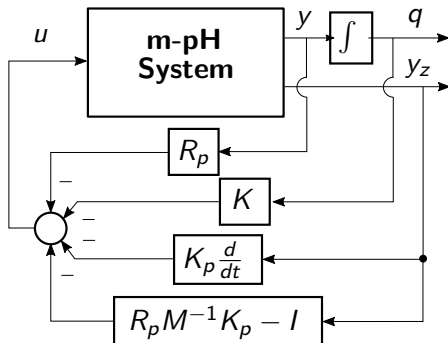
Consider the **new variable** q

- ① Mechanical interpretation: **displacement** of the finite dimensional pH system
- ② Dynamic defined as $\dot{q} = M^{-1}p$

Strong dissipation & position control: Asymptotic stability

Consider $K, R_p, K_p \in \mathbb{R}^{m \times m}$ diagonal and positive definite

$$u = -R_p M^{-1} p - Kq + (I - R_p M^{-1} K_p) C_1(\mathcal{H}z) - K_p \frac{d}{dt} (C_1(\mathcal{H}z))$$



Strong dissipation & position control: Asymptotic stability

Closed-loop operator, $x = [z \ p \ q]^T \in X = L_2([0, L], \mathbb{R}^n) \times \mathbb{R}^{2m}$

$$\begin{bmatrix} \dot{z} \\ \dot{p} \\ \dot{q} \end{bmatrix} = \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 & 0 \\ -R_p M^{-1} K_p C_1 \mathcal{H} - K_p \frac{d}{dt} C_1 \mathcal{H} & -R_p M^{-1} & -K \\ 0 & M^{-1} & 0 \end{bmatrix}}^{A_1} \begin{bmatrix} z \\ p \\ q \end{bmatrix}$$

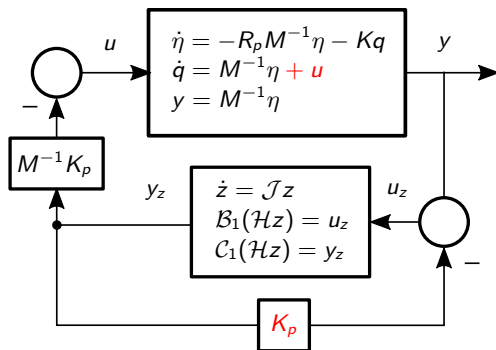
$$D(A_1) = \{x \in X \mid z \in H^1, \mathcal{B}_1(\mathcal{H}z) = M^{-1}p, \mathcal{B}_2(\mathcal{H}z) = 0\}$$

Change of variables $\eta = p + K_p C_1(\mathcal{H}z)$, $x = [z \ \eta \ q]^T$

$$\begin{bmatrix} \dot{z} \\ \dot{\eta} \\ \dot{q} \end{bmatrix} = \overbrace{\begin{bmatrix} P_1 \frac{\partial}{\partial \xi} \mathcal{H} + P_0 \mathcal{H} & 0 & 0 \\ 0 & -R_p M^{-1} & -K \\ -M^{-1} K_p C_1 \mathcal{H} & M^{-1} & 0 \end{bmatrix}}^A \begin{bmatrix} z \\ \eta \\ q \end{bmatrix}$$

$$D(A) = \{x \in X \mid z \in H^1, \mathcal{B}_1(\mathcal{H}z) = M^{-1}(\eta - K_p C_1(\mathcal{H}z)), \mathcal{B}_2(\mathcal{H}z) = 0\}$$

Strong dissipation & position control: Asymptotic stability



- 1 Not classical finite dimensional port-Hamiltonian system
- 2 Boundary dissipation on the infinite dimensional system

Strong dissipation & position control: Asymptotic stability

Theorem 3

Assume that the control parameters are selected such that $r_i^2 \geq 2m_i k_i$ $i \in \{1, \dots, m\}$, then the operator A generates a contraction C_0 -semigroup in the space X equipped with the weighted norm

$$\|x\|_{\Gamma}^2 = \langle x, x \rangle_{\Gamma} = \langle z, \mathcal{H}z \rangle_{L_2} + [\eta \ q] \begin{bmatrix} K^{-1}M^{-1}R_p K_p^{-1} & K_p^{-1} \\ K_p^{-1} & 2KMR_p^{-1}K_p^{-1} \end{bmatrix} \begin{bmatrix} \eta \\ q \end{bmatrix}$$

Moreover A has a compact resolvent.

\Rightarrow Pre-compactness of the solution set $\{x(t, x_0) \mid t \geq 0\}$

Strong dissipation & position control: Asymptotic stability

Theorem 4

Assume that the distributed parameter part of the system is **approximately observable** w.r.t. the output y_z and $\mathbf{z}_{eq} = \mathbf{0}$ is its only **equilibrium point**. If the control gains $k_{p,i}, r_i, k_i$ with $i = \{1, \dots, m\}$ are chosen such that $r_i^2 > 2m_i k_i, k_{p,i} > 0$, then the origin $x_{eq} = 0$ is a **globally asymptotically stable equilibrium**.

Strong dissipation & position control: Asymptotic stability

Theorem 4

Assume that the distributed parameter part of the system is **approximately observable** w.r.t. the output y_z and $z_{eq} = \mathbf{0}$ is its only **equilibrium point**. If the control gains $k_{p,i}, r_i, k_i$ with $i = \{1, \dots, m\}$ are chosen such that $r_i^2 > 2m_i k_i$, $k_{p,i} > 0$, then the origin $x_{eq} = 0$ is a **globally asymptotically stable equilibrium**.

Sketch of the Proof.

Define the candidate **Lyapunov function** $V(x) = \frac{1}{2} \langle x, x \rangle_\Gamma$

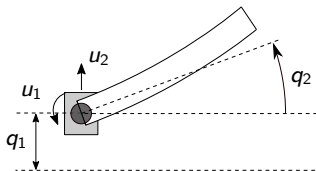
$$\dot{V} = dVAx = \langle x, Ax \rangle_\Gamma.$$

Using approximate observability and $z_{eq} = 0$, the largest invariant subset of $\mathbf{S}_0 = \{x \in X \mid \dot{V}(x) = 0\}$ corresponds to $\mathbf{S} = \{0\}$. With the LaSalle's invariance principle, $\lim_{t \rightarrow \infty} \|x(t)\| = 0$.

Comparison of the control laws

Comparison of the control laws

Example: Flexible beam clamped on a translating rotating inertia

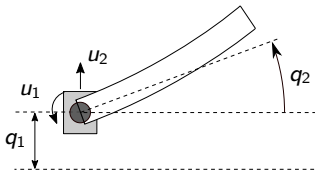


Control objectives:

- 1 Stabilise the system starting from any initial condition.

Comparison of the control laws

Example: Flexible beam clamped on a translating rotating inertia

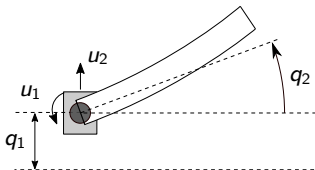


Control objectives:

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Comparison of the control laws

Example: Flexible beam clamped on a translating rotating inertia

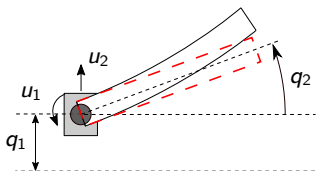


Control objectives:

- 1 Stabilise the flexible part of the mechanism starting from an arbitrary initial condition.
- 2 Stabilise the flexible part of the mechanism and position the overall mechanism in a desired configuration.

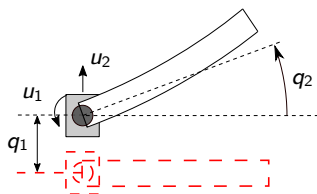
Comparison of the control laws

Strong dissipation



- 1 Exponential stabilisation of the flexible part of the mechanism
- 2 We do not have any control on the Mechanism's position

Strong dissipation & position control



- 1 Stabilisation of the mechanism in the desired configuration
- 2 We do not know the convergence rate

Conclusions, ConFlex fellowship & future plans

Conclusions & future research

Conclusions

- ① Strong dissipation control: lack of direct control on the PDE's boundaries, Exponential stability, lack of position control.
- ② Strong dissipation & position control: lack of direct control on the PDE's boundaries, Asymptotic stability, position control.

Future research

Modify the “Strong dissipation & position control law” such to obtain exponential stability.

Publications

Journal papers

- A. Mattioni, Y. Wu, Y. Le Gorrec. "Infinite dimensional model of a double flexible-link manipulator: The Port-Hamiltonian approach". Applied Mathematical Modelling, Elsevier. (2020)
- A. Mattioni, Y. Wu, H. Ramirez, Y. Le Gorrec, A. Macchelli. "Modelling and control of an IPMC actuated flexible structure: A lumped port Hamiltonian approach". Control Engineering Practice, Elsevier. (2020)
- A. Mattioni, Y. Wu, Y. Le Gorrec, H. Zwart. "Stabilization of a Class of Mixed ODE-PDE port-Hamiltonian Systems with Strong Dissipation Feedback". Major revision in Automatica.
- A. Mattioni, Y. Wu, Y. Le Gorrec. "Asymptotic stability of a flexible beam clamped on a rotating inertia entering in contact with the external environment". In preparation

Conference papers

- A. Mattioni, J. Toledo, Y. Le Gorrec. "Observer based nonlinear control of a rotating flexible beam". IFAC 2020 World Congress (2020).
- A. Mattioni, Y. Wu, Y. Le Gorrec, H. Zwart. "Stabilisation of a rotating beam clamped on a moving inertia with strong dissipation feedback". Control and Decision Conference (2020).
- A. Mattioni, Y. Wu, Y. Le Gorrec. "Modelling, Control and Stability Analysis of Flexible Rotating Beam's Impacts During Contact Scenario". American Control Conference (2021).
- A. Mattioni, Y. Wu, Y. Le Gorrec. "Exponential stabilization of a clamped Timoshenko beam with actuation on a tip mass". Control and Decision Conference (2021).

Secondments, Trainings & Outreach activities

Secondments

- Thales Alenia Space, Cannes, France. March - April 2019
- University of Wuppertal, Wuppertal, Germany. June 2019
- University of Twente, Enschede, The Netherlands. October - November 2019

Training and Outreach activities

- “Spring School on Theory and Applications of Port-Hamiltonian Systems”, Munich, Germany. April 2019.
- 21st IFAC World Congress, Virtual. July 2020.
- 59st Control and Decision Conference, December 2020.
- 2021 American Control Conference, May 2021.

PhD & Future plans

PhD status

I got the PhD degree on “Automatic Control” by the University of Bourgogne Franche Comté on the 23rd April 2021.

Future plans

Post-doc position at the Gipsa-lab of the University of Grenoble on the subject: “Reinforcement Learning control of nonlinear PDE”.
In collaboration with Christophe Prieur and Paolo Frasca.

Thanks for your attention!